

32.2 A skylight is comprised of two layers of $\frac{1}{2}$ inch thick laminated glass with a 1 inch air gap $\left(R = 0.5 \frac{\text{hr} \cdot \text{ft}^2 \cdot ^\circ\text{F}}{\text{Btu}}\right)$ in between. The inside and outside film coefficients are $1.5 \frac{\text{Btu}}{\text{hr} \cdot \text{ft}^2 \cdot ^\circ\text{F}}$ and $2.0 \frac{\text{Btu}}{\text{hr} \cdot \text{ft}^2 \cdot ^\circ\text{F}}$, respectively. The outside design temperature is 10°F and the inside conditions are 70°F with 60% relative humidity. What is the maximum allowable thermal conductivity of the glass to prevent condensate from forming on the inside of the skylight?

- A. $0.04 \frac{\text{Btu}}{\text{hr} \cdot \text{ft} \cdot ^\circ\text{F}}$
- B. $0.08 \frac{\text{Btu}}{\text{hr} \cdot \text{ft} \cdot ^\circ\text{F}}$
- C. $0.2 \frac{\text{Btu}}{\text{hr} \cdot \text{ft} \cdot ^\circ\text{F}}$
- D. $0.9 \frac{\text{Btu}}{\text{hr} \cdot \text{ft} \cdot ^\circ\text{F}}$

The inside conditions are known. Determine the **Dew Point** temperature using the **Psychrometric Chart**. This is the minimum temperature on the inside surface of the glass before which condensate will form.

$$T_{i,db} = 70^\circ\text{F}$$

$$\phi_i = 60\%$$

$$T_{i,dp} = 55.6^\circ\text{F}$$

Let T_i be the inside temperature.

Let A denote the inner surface of the skylight.

Let B denote the interface between the inner glass and the air gap.

Let C denote the interface between the outer glass and the air gap.

Let D denote the outer surface of the skylight.

Let T_o be the outside temperature.

Review the Reference Handbook section on **Conduction**. Treat the skylight as a **Composite Plane Wall**, where the total resistance, R_{total} , can be expressed for this particular situation as:

$$R_{total} = \frac{1}{h_i} + \frac{L}{k} + R + \frac{L}{k} + \frac{1}{h_o}$$

where h_i and h_o are the inside and outside film coefficients, respectively, the $\frac{L}{k}$ terms correspond to conduction through the thickness L of the both sheets of glass having thermal conductivity k , and R represents the resistance of the air gap, which is given. Substitute and simplify to derive an expression for the total resistance, R_{total} , as a function of thermal conductivity, k . Make sure units for each term are in alignment.

$$R_{total} = \frac{1}{1.5 \frac{\text{Btu}}{\text{hr} \cdot \text{ft}^2 \cdot ^\circ\text{F}}} + \frac{\left(\frac{1}{2}\text{in}\right) \left(\frac{1\text{ft}}{12\text{in}}\right)}{k} + 0.5 \frac{\text{hr} \cdot \text{ft}^2 \cdot ^\circ\text{F}}{\text{Btu}} + \frac{\left(\frac{1}{2}\text{in}\right) \left(\frac{1\text{ft}}{12\text{in}}\right)}{k} + \frac{1}{2.0 \frac{\text{Btu}}{\text{hr} \cdot \text{ft}^2 \cdot ^\circ\text{F}}}$$

$$R_{total} = 1.67 \frac{hr \cdot ft^2 \cdot ^\circ F}{Btu} + \frac{1ft}{12(k)}$$

The inside temperature will be minimized when the heat loss through the skylight is maximized, i.e. when the heat flux through the “composite wall” is large. Consider the **Convection** between the inside ambient space at temperature T_i , and the inside of the skylight at surface A. The heat transfer by convection is given by:

$$\dot{Q} = h_i A \Delta T$$

Since the area is unknown, divide by area and work with the heat transfer *per unit area*, or heat flux:

$$\dot{q} = h_i \Delta T$$

$$\dot{q}_{max} = \left(1.5 \frac{Btu}{hr \cdot ft^2 \cdot ^\circ F} \right) (70^\circ F - 55.6^\circ F) = 21.6 \frac{Btu}{hr \cdot ft^2}$$

Considering the entire wall again, find the overall heat transfer coefficient, $U_{overall}$, inclusive of all layers and resistances by assuming the maximum heat flux through the resistance of all layers. The heat flow through the wall can be represented as:

$$\dot{Q} = U A \Delta T$$

Again, we need not be concerned with the area. Divide by area and work with the heat flux:

$$\frac{\dot{Q}}{A} = \dot{q} = U \Delta T$$

Assume the maximum heat flux as previously calculated, and use the inside and outside temperature given to calculate U :

$$U = \frac{\dot{q}_{max}}{\Delta T} = \frac{21.6 \frac{Btu}{hr \cdot ft^2}}{70^\circ F - 10^\circ F} = 0.36 \frac{Btu}{hr \cdot ft^2 \cdot ^\circ F}$$

The Total Resistance, R_{total} , is the inverse of the Overall Coefficient of Heat Transfer, U .

$$U = \frac{1}{R_{total}} \rightarrow R_{total} = \frac{1}{U}$$

$$R_{total} = \frac{1}{0.36 \frac{Btu}{hr \cdot ft^2 \cdot ^\circ F}} = 2.77 \frac{hr \cdot ft^2 \cdot ^\circ F}{Btu}$$

Set the calculated value for total resistance equal to the expression for total resistance derived for the composite wall, and solve for the thermal conductivity:

$$R_{total} = 1.67 \frac{hr \cdot ft^2 \cdot ^\circ F}{Btu} + \frac{1ft}{12(k)} = 2.77 \frac{hr \cdot ft^2 \cdot ^\circ F}{Btu}$$

$$\frac{1ft}{12(k)} = 1.11 \frac{hr \cdot ft^2 \cdot ^\circ F}{Btu}$$

$$k = \frac{\left(\frac{1ft}{12}\right)}{\left(1.11 \frac{hr \cdot ft^2 \cdot ^\circ F}{Btu}\right)} = .075 \frac{Btu}{hr \cdot ft \cdot ^\circ F}$$

Answer B